# **Quenching and Suppression of Limit Cycles in 3x3 Nonlinear Systems**

KARTIK CHANDRA PATRA<sup>1\*</sup>, ASUTOSH PATNAIK<sup>2</sup> <sup>1,2</sup>Department of Electrical Engineering C. V. Raman Global University Bhubaneswar, Odisha, PIN - 752054 INDIA \*Corresponding Author \*ORCID ID: 0000-0002-4693-4883

Abstract: - For several decades, the importance and weight-age of prediction of nonlinear self-sustained oscillations or Limit Cycles (LC) and their quenching by signal stabilization have been discussed which is confined to Single Input and Single Output (SISO) system. However, for the last five to six decades, the analysis of 2x2 Multi Input and Multi Output (MIMO) Nonlinear Systems gained importance in which a lot of literature available. In recent days few literatures are available which addresses the exhibition of LC and their quenching/suppression in 3x3 MIMO Nonlinear systems. Poor performances in many cases like Load Frequency Control (LFC) in multi area power system, speed and position control in robotics, automation industry and other occasions have been observed which draws attention of Researchers. The complexity involved, in implicit nonmemory type and memory type nonlinearities, it is extremely difficult to formulate the problem in particular for 3x3 systems. Under this circumstance, the harmonic linearization/ harmonic balance reduces the complexity considerably. Still the analytical expressions are so complex which loses the insight into the problem particularly for memory type nonlinearity in 3x3 system. Hence in the present work a novel graphical method has been developed for prediction of limit cycling oscillations in a 3x3 nonlinear system. The quenching of such LC using signal stabilization technique using deterministic (Sinusoidal) and random (Gaussian) signals has been explored. Suppression LC using pole placement technique through arbitrary selection and optimal selection of feedback Gain Matrix K with complete state controllability condition and Riccati Equation respectively. The method is made further simpler assuming a 3x3 system exhibits the LC predominantly at a single frequency, which facilitates clear insight into the problem and its solution.

The proposed techniques are well illustrated with example and validated/substantiated by digital simulation (a developed program using MATLAB codes) and use of SIMULINK Tool Box of MATLAB software.

The Signal stabilization with Random (Gaussian) Signals and Suppression LC with optimal selection of state feedback matrix K using Riccati Equation for 3x3 nonlinear systems have never been discussed elsewhere and hence it claims originality and novelty.

The present work has the brighter future scope of:

i. Adapting the Techniques like Signal Stabilization and Suppression LC for 3x3 or higher dimensional nonlinear systems through an exhaustive analysis.

ii. Analytical/Mathematical method may also be developed for signal stabilization using both deterministic and random signals based on Dual Input Describing function (DIDF) and Random Input Describing Function (RIDF) respectively.

iii. The phenomena of Synchronization and De-synchronization can be observed/identified analytically using Incremental Input Describing Function (IDF), which can also be validated by digital simulations.

*Key-Words:* - Limit Cycles, Describing function, 3x3 non-linear systems, Pole placement technique, Suppression limit cycle, signal stabilization.

Received: March 11, 2024. Revised: August 28, 2024. Accepted: September 21, 2024. Published: November 19, 2024.

# **1** Introduction

It has been observed for years long the importance and weightage on self-sustained oscillations or nonlinear oscillations or limit cycles (LC) [1], [2], [3], [4], [5].

For last several decades, the analysis of 2x2 multivariable nonlinear systems drawn attention of the researchers and good number of literature is available [6], [7], [8], [9], [10], [11], [12], [13], [14], [15], [16], [17], [18], [19], [20], [21], [22], [23], [24], [25], [26], [27], [28], [29], [30], [31], [32], [33], [34], [35], [36], [37], [38], [39], [40], [41], [42], [43], [44], [45], [46], cover this area of research. The prediction of LC in 2 x 2 system, in means of increasing the reliability of the describing function (DF) are well established [4], [5], [10], [13], [16], [23], [47], [48] and others used harmonic linearization/ harmonic balance, [13], [27], [31], [49].

In the event of existence of limit cycling oscillations, the possibility, of quenching the sustained oscillations using the method of signal stabilization has been investigated, [5], [28], [29], [47], [48], in 2X2 nonlinear systems with non-memory type nonlinear elements and in memory type nonlinear elements in [37] using deterministic signals whereas the same has been addressed with Gaussian signals [52].

Prediction and suppression of limit cycling oscillations in 2 x 2 memory type nonlinear systems using arbitrary pole placement has been discussed in [30], [41], [42], [50].

The present work follows the dynamics of general 3X3 nonlinear systems shown in Fig. 2, Fig.3 [6], which is an equivalent representation of the general multivariable system of Fig.1[26].

Having realized the importance of quenching/suppression of limit cycle oscillations the present work first establishes the exhibitions of limit cycles in 3X3 nonlinear systems following the similar procedure as depicted/illustrated [6].

# 2. Graphical Method of prediction of LC in a general 3x3 Nonlinear Systems

In order to avoid complexity, involved in this structure a graphical method is developed for prediction of limit cycles in 3x3 nonlinear systems [6], [53].

### 2.1 Graphical Method

Consider a system of Fig.1, a class of 3x3 nonlinear systems for simplicity it is assumed that the whole 3x3 system exhibits the LC predominantly of a single frequency sinusoid and harmonic linearization/harmonic balance leading to use of describing function methods have been opted.

The normalized phase diagrams [44] are drawn for 3x3 systems with three combinations such as:

Combination 1: For subsystems S1, S2 & S3: C1 (+ve), C2 (-ve) and C3 (+ve)

Combination 2: For subsystems S3, S2 & S1: C2 (+ve), C3 (-ve) and C1 (+ve).

Combination 3: For subsystems S1, S3 & S2: C3 (+ve), C1 (-ve) and C2 (+ve).

Example: Used for illustration of procedures of Normalized phase diagrams.

The linear elements are represented by  $G_1(s) = \frac{2}{s(s+1)^2}$ ;  $G_2(s) = \frac{1}{s(s+4)}$ ;  $G_3(s) = \frac{1}{s(s+2)}$  and Nonlinear elements are taken, Ideal relays as shown in Fig.2.



Fig. 1: A class of 3x3 multivariable nonlinear systems



Fig. 2: All Ideal Relays

Assuming harmonic linearization these nonlinear elements be equivalently can represented by their describing functions which are real functions in these two examples and do not contribute any phase angles to the system. Hence the phase angles of the system are due to linear functions,  $G_1(s)$ ,  $G_2(s)$ ,  $G_3(s)$  which are complex functions of complex variable s, the Laplace operator. It may be noted that for frequency response, input is sinusoidal and outputs are steady state values considered, so that s (Laplace Operator) is replaced by  $j\omega[6]$ .

X<sub>1</sub>, X<sub>2</sub> & X<sub>3</sub> are the amplitudes of respective sinusoidal inputs to the nonlinear elements. C<sub>1</sub>, C<sub>2</sub> & C<sub>3</sub> are the amplitudes of sinusoidal output of subsystems S<sub>1</sub>, S<sub>2</sub> & S<sub>3</sub> respectively. G<sub>1</sub>, G<sub>2</sub> & G<sub>3</sub> are the magnitudes/absolute values of linear elements represented by their transfer functions of subsystems S<sub>1</sub>, S<sub>2</sub> & S<sub>3</sub> respectively and N<sub>1</sub>, N<sub>2</sub> & N<sub>3</sub> are the magnitudes/absolute values of linear elements represented by their describing functions of subsystems S<sub>1</sub>, S<sub>2</sub> & S<sub>3</sub> respectively.

$$\theta_{L1} = \operatorname{Arg.} (\mathbf{G_1} (j\omega)) = -90 - 2\tan^{-1}(\omega):$$
  

$$\theta_{L2} = \operatorname{Arg.} (\mathbf{G_2} (j\omega)) = -90 - \tan^{-1}(\frac{\omega}{4}):$$
  

$$\theta_{L3} = \operatorname{Arg.} (\mathbf{G_3} (j\omega)) = -90 - \tan^{-1}(\frac{\omega}{2}):$$
  

$$N_2 = (11 - 3\omega^2)\omega^2 \pm$$
  

$$\sqrt{(11 - 3\omega^2)\omega^4 - 8(\omega^2 + 16)(1 - \omega^2)^2\omega^2}$$
  

$$\dots (1),$$
  

$$N_1 = \frac{(\omega^2 - 1)}{8} N_2 + \frac{9\omega^2 - \omega^4}{8} \qquad \dots (2),$$

$$\frac{X_{1}}{X_{2}} \frac{(1+\omega^{2})\sqrt{[\omega^{2}(\omega^{2}+16-2N_{2})+N_{2}^{2}]}}{2N_{1}\sqrt{\omega^{2}+16}} \qquad \cdots (3),$$

With reference to Fig. 3(a),



$$\frac{X_1}{X_2} = \frac{BD_i}{AD_i} = \sqrt{\frac{(1-u_i)^2 + (u_i)^2}{(1+u_i)^2 + (u_i)^2}} \qquad \dots (4)$$

For a fixed value of  $\omega$  the combinations of subsystems 1, 2, and 3, Normalised Phase Diagrams are shown in Figure 3(a), (b), and (c) respectively. However, *any one* of these combinations can be used for the determination of limit cycling conditions and the related quantities of interest.

FIG.3 (a): Normalised phase diagram with  $C_1$ ,  $C_2$  &  $C_3$  for the combination 1, where  $C_1$  (+ve),  $C_2$  (-ve) and  $C_3$  (+ve).



FIG.3 (b): Normalised phase diagram with  $C_1$ ,  $C_2$  &  $C_3$  for the combination 2, where  $C_2$  (+ve),  $C_3$  (-ve) and  $C_1$  (+ve).

FIG.3 (c): Normalised phase diagram with  $C_1$ ,  $C_2$  &  $C_3$  for the combination 3,  $C_3$  (+ve),  $C_1$  (-ve) and  $C_2$  (+ve).

With reference to a normalized phase diagram [44], the phase representing X<sub>2</sub> would lie along a straight line drawn at an angle  $\theta_{L2}$  with the phase C<sub>2</sub> (C<sub>2</sub> = -R<sub>1</sub>). The intersections of this straight line with the circle drawn with respect to  $\theta_{L1}$  would represent possible self-oscillations. The concept has been extended for 3 x 3 as: (i) Consider Fig. 3(a) the phase representing  $X_2$  and  $X_3$  would lie along straight lines drawn at angles  $\theta_{L2}$  and  $\Theta_{L3}$  with the phase  $C_2$  ( $C_2 = - R_1$ ) and  $C_3$  ( $C_3=R_1$ ) respectively. The intersections of these straight lines with the circle drawn with respect to  $\theta_{L1}$  would represent possible self-oscillations.

(ii) Consider Fig. 3(b), the phase representing  $X_3$  and  $X_1$  would lie along straight lines drawn at angles  $\theta_{L3}$  and  $\theta_{L1}$  with the phase  $C_3$  ( $C_3$ = -  $R_2$ ) and  $C_1$  ( $C_1$ = $R_2$ ) respectively. The intersections of these straight lines with the circle drawn with respect to  $\theta_{L2}$  would represent possible self-oscillations.

(iii) Consider Fig. 3(c) the phase  $X_1$  and  $X_2$  would lie along straight lines drawn at angles  $\theta_{L1}$  and  $\theta_{L2}$  with phase  $C_1(C_1=-R_3)$  and  $C_2(C_2=R_3)$  respectively. The intersections of these straight lines with the circle drawn with respect to  $\theta_{L3}$  would represent possible self-oscillations.

Table 1: Shows the  $\theta_{L1}$ ,  $\theta_{L2}$ ,  $\theta_{L3}$ , r (radius), and the intersection points of the straight lines and circle for combination 1 corresponding to the *example*. It may

be noted that Table 1: Contains  $\frac{X_1}{X_2}$  obtained from

Eqn.3 and Eqn.4 are matched at a limit cycling frequency.

# 2.2 Digital Simulation

The Example is revisited:

A program has been developed [6] with the use of MATLAB code for digital simulation.

The equivalent canonical form of Fig. 1 for the example is shown in Fig. 4(a) and digital representation is shown in Fig. 4(b) respectively.

Numerical results obtained from different methods are compared in Table 2 for the example.

The results/images for the example obtained from digital simulation (using the developed program) and that of obtained using SIMULINK Toolbox of MATLAB software are shown in Fig. 5 for comparison.



Fig. 4(a): Equivalent Canonical form of Fig.1 for the Example



Fig. 4(b): The Digital representation of Fig.1 for the Example

ω	θ <sub>L1</sub>	$\theta_{L2}$	$\theta_{L3}$	r	X <sub>1</sub> /X 2 from eqn. 3	X <sub>1</sub> /X <sub>2</sub> from eqn. 4	Normalized Phase Diagrams	Remark	
0.600	-151.93	-98.531	-106.7	-0.55257	-	-		No intersection of straight lines and circle	
0.650	-156.05	-99.23	-108	0.58256	-	-		No intersection of straight lines and circle	
0.700	-159.98	-99.926	-109.29	-2.128	-	-		No intersection of straight lines and circle	
0.701	-160.06	-99.94	-109.32	-3.1323	1.0	1.02 (matched )		The intersection of st. lines & circle found: Confirms the occurrence of limit cycles $\omega$ =0.701, C1 = OD2 = 6 C2 = 1 C3 = 1 X1=BD2=6.08 X2=AD2=6.08 X3=B'D2= 6.32	
0.750	-163.74	-100.62	-110.56	-1.3583	_	-		No intersection of straight lines and circle	

Table 1: Shows the  $\theta_{L1}$ ,  $\theta_{L2}$ ,  $\theta_{L3}$ , r (radius), and the intersection points of the straight lines and circles for combination 1 corresponding to the Example (with reference to Fig. 3(*a*)).



Fig. 5: Results/Images from digital simulation and SIMULINK for  $C_1$ ,  $C_2$ ,  $C_3$ ,  $X_1$ ,  $X_2$  and  $X_3$  of the Example (relay type nonlinearities).

Table 2 Results obtained using different methodscorresponding to Ideal Relay Example

Sl. No	Methods	C <sub>1</sub>	<b>C</b> <sub>2</sub>	<b>C</b> <sub>3</sub>	<b>X</b> <sub>1</sub>	<b>X</b> <sub>2</sub>	<b>X</b> <sub>3</sub>	ω
1	Graphical	6.0	1.0	1.0	6.08	6.08	6.32	0.701
2	Digital Simulation (developed program)	4.83	0.74	0.95	4.72	4.91	5.23	0.70
3	Using SIMULINK TOOL BOX OF MATLAB	5.95	1.01	0.96	4.84	5.12	5.62	0.70

# 3. Signal Stabilization in 3x3 Nonlinear System

The System exhibits limit cycles (LC) in the autonomous state, the possibility of quenching the LC by injecting a suitable high frequency signal, preferably, at least 10 times of the limit cycling frequency.

# **3.1 Using Deterministic Signal**

The forced oscillations can be realised by feeding deterministic or random signals of high frequency, at least greater than 10 times the limit cycling frequency at any one /all input points of the subsystems  $S_1$ ,  $S_2$ ,  $S_3$ .

If the amplitude B of the high frequency signal is gradually increased, the system would exhibit complex oscillations before the synchronization takes place. On the reverse operation, if the amplitude B is gradually reduced at certain value of B the self-oscillations i.e. the Limit cycle would reappear and the system would exhibit complex oscillations again which can be called the desynchronisation. The phenomena of synchronization and de-synchronization can be observed / identified analytically using Incremental Input Describing function (IDF) [44].

However, the forced oscillation can also be analysed using the Equivalent Gain/Dual input Describing Function (DIDF) [44] in case of a deterministic forcing signal in particular with a sinusoidal signal. Taking the second option i.e. all three inputs are same as B sin $\omega_{f}$ t at 3 input points U<sub>1</sub>, U<sub>2</sub>, & U<sub>3</sub>, shown in Fig.6. Amplitude B is gradually increased, the frequency of selfoscillation,  $\omega_{s}$  would gradually change, the system will synchronize to forcing frequency i.e. the selfoscillation would be quenched and the system would



exhibit forced oscillations at frequency  $\omega_{f}$ .

Fig 6: Equivalent System of Fig. 1 for forced oscillations (Signal Stabilization) with deterministic signal for the Example.

The results/images from digital simulation for signal stabilization with deterministic (sinusoidal signal) for the Example shown in Fig.7.



Fig. 7: Forced Oscillations by Signal Stabilization with deterministic signal for the Example Forcing Signal U =  $5\sin\omega_f t(\omega_f = 7.5 \text{ rad} / \text{sec})$ 

#### 3.2 Using Gaussian signal:

The forced oscillation is analysed with Equivalent Gain (similar to DIDF - Random Input Describing Function) (RIDF) in case of the Random Signals in particular with Gaussian Signals [52].

Consider the *Example*. The system is exhibiting LC under autonomous state, a Gaussian signal with specified *mean* and *variance* is injected at  $U_1$ ,  $U_2$  &  $U_3$  of subsystems for stabilizing the system / quenching the self-sustained oscillations. At a suitable value of mean ( $\mu$ ) and variance ( $\phi$ ), the self-sustained oscillations are vanished/the system is synchronised to high frequency forcing input.

The results/images are shown in Fig. 8, which is obtained from digital simulation by signal stabilization with Gaussian signals for the *Example* replacing B sin  $\omega_f t$  with a suitable random signals in Fig.7.



Fig. 8: Forced Oscillations by signal stabilization with Gaussian Signal of mean 50 and variance 0.05 for the Example

# 4. Suppression of limit cycle in 3x3 nonlinear system using pole placement technique

The System of the *Example* exhibits Limit Cycles which can be suppressed by pole placement technique [46]. The closed loop poles or Eigen values of the closed loop systems can be placed at the desired location through state feedback using an appropriate feedback gain matrix K [ $k_1$ ,  $k_2$ ,  $k_3$ ]. Necessary and sufficient condition for arbitrary pole placement is that the system be completely state controllable [46]. This can also be done by optimal

selection of feedback gain matrix K using Riccati Equation [46].

# 4.1 Suppression of Limit Cycles in 3x3 Nonlinear system using arbitrary Pole Placement by state feedback:

Pole placement technique by state feedback is done by determining the Eigen values or poles of the system. These Eigen values cause the limit cycles in the system, and as the complete removal of these selfoscillations may not be possible, the location of the poles must be changed from its original position so as to bring about suppression of the limit cycle. The most general multivariable nonlinear system [53] is shown in Fig. 9 (a). For existence of limit cycles, an autonomous system (input U=0) Fig. 9(a) can be represented in simplified form as shown in Fig. 9(b). Making use of the first harmonic linearization of the nonlinear elements, the matrix equation for the system of Fig. 9(b) can be expressed as

X = -HC, where C = GN(x) X. Hence,

$$X = -HGN(x) = AX \qquad \cdots \qquad (5)$$

Where, A = -HGN(x)



Fig. 9(a): Block diagram representation of a most general nonlinear multivariable system



Fig. 9(b): Equivalent of the system of Fig. 9 (a) with input U=0

Realizing Eqn. (5) as a transformation of the vector X onto itself, it is noted that for a limit cycle to exist the following two conditions should be satisfied, [6], [53]:

- (i) For every non-trivial solution of X, the matrix A must has an Eigen value  $\lambda$  equal to unity, and
- (ii) The Eigen vector of "A" corresponding to this unity Eigen value must be coincident with X.

In order to suppress the limit cycles, arbitrary pole placements may be possible if the system is completely state controllable [46].

The controllability matrix  $S = [B AB A^2 B ... ...]$ 

(6)

Where,

....

$$\begin{split} \mathbf{A} &= \begin{bmatrix} -\mathbf{N}_{1}\mathbf{G}_{1} & -\mathbf{N}_{2}\mathbf{G}_{2} & \mathbf{N}_{3}\mathbf{G}_{3} \\ \mathbf{N}_{1}\mathbf{G}_{1} & -\mathbf{N}_{2}\mathbf{G}_{2} & -\mathbf{N}_{3}\mathbf{G}_{3} \\ -\mathbf{N}_{1}\mathbf{G}_{1} & \mathbf{N}_{2}\mathbf{G}_{2} & -\mathbf{N}_{3}\mathbf{G}_{3} \end{bmatrix}; \mathbf{B} = \begin{bmatrix} \mathbf{0} \\ \mathbf{0} \\ \mathbf{1} \end{bmatrix}; \\ \mathbf{H} &= \begin{bmatrix} \mathbf{1} & \mathbf{1} & -\mathbf{1} \\ -\mathbf{1} & \mathbf{1} & \mathbf{1} \\ \mathbf{1} & -\mathbf{1} & \mathbf{1} \end{bmatrix}; \\ \mathbf{G}(\omega) &= \begin{bmatrix} \mathbf{G}_{1}(\omega) & \mathbf{0} & \mathbf{0} \\ \mathbf{0} & \mathbf{G}_{2}(\omega) & \mathbf{0} \\ \mathbf{0} & \mathbf{0} & \mathbf{G}_{3}(\omega) \end{bmatrix}; \\ \mathbf{N}(\mathbf{X}) &= \begin{bmatrix} \mathbf{N}_{1}(\mathbf{X}_{1}) & \mathbf{0} & \mathbf{0} \\ \mathbf{0} & \mathbf{N}_{2}(\mathbf{X}_{2}) & \mathbf{0} \\ \mathbf{0} & \mathbf{0} & \mathbf{N}_{3}(\mathbf{X}_{3}) \end{bmatrix}; \\ \mathbf{X} &= \begin{bmatrix} \mathbf{X}_{1} \\ \mathbf{X}_{2} \\ \mathbf{X}_{3} \end{bmatrix}; \ \mathbf{C} = \begin{bmatrix} \mathbf{C}_{1} \\ \mathbf{C}_{2} \\ \mathbf{C}_{3} \end{bmatrix} \end{split}$$

From Table 1 for the Example,

 $\omega = 0.701 \text{ radian/sec}, X_1 = 6.08, X_2 = 6.08, X_3 = 6.32$  $N_1(X_1) = \frac{4M_1}{\pi X_1} = \frac{4x2}{\pi x 6.08} = 0.419; N_2(X_2) = \frac{4M_2}{\pi X_2} = \frac{4x1.5}{\pi x 6.08}$  $= 0.314, N_3(X_3) = \frac{4M_3}{\pi X_3} = \frac{4x1}{\pi x 6.32} = 0.202$ 

$$|G_1(j\omega)| = \frac{2}{\sqrt{(\omega - \omega^3)^2 + (2\omega^2)^2}} = \frac{2}{\omega(\omega^2 + 1)} = 1.913$$

$$|G2(j\omega)| = \frac{1}{\sqrt{(\omega^2)^2 + (4\omega)^2}} = \frac{1}{\omega\sqrt{16 + \omega^2}} = 0.351$$

$$|\mathbf{G}_3| = \frac{1}{\omega \sqrt{\omega^2 + 4}} = 0.673$$
  
On substitution of the numerical values:

$$-N_1G_1 = -0.419 \text{ X } 1.913 = -0.802,$$

$$-N_2G_2 = -0.314 \times 0.351 = -0.110$$

$$-N_3G_3 = -0.202 \times 0.673 = -0.136$$

$$A = \begin{bmatrix} -0.802 & -0.110 & 0.136 \\ 0.802 & -0.110 & -0.136 \\ -0.802 & 0.110 & -0.136 \end{bmatrix} ; AB = \begin{bmatrix} -0.802 & -0.110 & 0.136 \\ 0.802 & -0.110 & -0.136 \\ -0.802 & 0.110 & -0.136 \end{bmatrix} \begin{bmatrix} 0 \\ 0 \\ 1 \end{bmatrix} = \begin{bmatrix} 0.136 \\ -0.136 \\ -0.136 \\ -0.136 \end{bmatrix};$$
$$A^{2}B = \begin{bmatrix} -0.802 & -0.110 & 0.136 \\ 0.802 & -0.110 & -0.136 \\ -0.802 & 0.110 & -0.136 \\ -0.136 \end{bmatrix} \begin{bmatrix} 0.136 \\ -0.136 \\ -0.136 \\ -0.136 \end{bmatrix} = \begin{bmatrix} -0.1125 \\ 0 & -0.136 \\ 0.1425 \\ -0.1055 \end{bmatrix}$$
Hence S = 
$$\begin{bmatrix} 0 & 0.136 & -0.1125 \\ 0 & -0.136 & 0.1425 \\ 1 & -0.136 & -0.1055 \end{bmatrix} = 0.0215 \neq 0 (The$$
system is completely state controllable)

Hence arbitrary pole placement is possible [46]

$$\frac{d}{dt}[x(t)] = AX + Bu \qquad \cdots \qquad (7)$$

The system under autonomous state is represented as shown in Fig. 10.



Fig. 10: A system with state feedback Consider Fig. 10: The control law u = -KX

Where  $K = \begin{bmatrix} k_1 & k_2 & k_3 \end{bmatrix}$  is the feedback matrix. Replacing K in Eqn. (7) by Eqn. (8), we get,

$$\frac{d}{dt}[x(t)] = (A-BK) X \quad \cdots \quad (9)$$

Substituting the values of A, B and K, we get: The Characteristic Equation as

$$\llbracket \lambda I - (A - BK) \rrbracket = 0 \text{ or }$$

(8)

$$\begin{bmatrix} (\lambda + N_1G_1) & N_2G_2 & -N_3G_3 \\ -N_1G_1 & (\lambda + N_2G_2) & N_3G_3 \\ N_1G_1 + k_1 & -N_2G_2 - k_2 & \lambda + N_3G_3 + k_3 \end{bmatrix} = \\ (\lambda + N_1G_1) \begin{vmatrix} \lambda + N_2G_2 & N_3G_3 \\ -N_2G_2 & \lambda + N_3G_3 + k_3 \end{vmatrix} \\ -N_2G_2 \begin{vmatrix} -N_1G_1 & N_3G_3 \\ N_1G_1 + k_1 & \lambda + N_3G_3 + k_3 \end{vmatrix} - N_3G_3 \\ \begin{vmatrix} -N_1G_1 & (\lambda + N_2G_2) \\ N_1G_1 + k_1 & -N_2G_2 - k_2 \end{vmatrix} = \\ = \lambda^3 + \lambda^2 (N_1G_1 + N_2G_2 + N_3G_3 + k_3) + \lambda (2N_1N_2G_1G_2 + 2) \\ N_1N_3G_1G_3 + 2N_2N_3G_2G_3 + k_1N_3G_3 + k_3N_1G_1 + \\ k_3N_2G_2 + k_2N_3G_3) + (4N_1N_2N_3G_1G_2G_3 + ) \end{bmatrix}$$

 $2k_3N_1N_2G_1G_2+2k_1N_2N_3G_2G_3 = 0 \cdots \cdots (10)$  (Ch. Equation)

On substitution of the values of  $N_1, G_1, N_2, G_2$  and

**N<sub>3</sub>**, **G<sub>3</sub>** in Eqn. (10), we get,

 $\lambda^{3} + \lambda^{2} (0.136 + 0.11 + 0.802 + k_{3}) + \lambda \{$ 

 $0.177+0.218+0.030+k_1 \times 0.136+k_3(0.80+0.11)+$ 

k<sub>2</sub>x 0.136}+(0.048+0.522+k<sub>1</sub>x0.03)=0 Or

 $\lambda^{3} + \lambda^{2} (1.048 + k_{3}) + \lambda (0.425 + k_{1}x_{0.136})$ 

 $+k_2 \times 0.136 + k_3 \times 0.91) + (0.57 + k_1 \times 0.03) = 0 \cdots$ (11)

If the poles are selected arbitrarily at  $\lambda_1, \lambda_2, \lambda_3 = -1, -1 \& -2$  respectively, the

characteristic equation becomes:

$$(\lambda + 1)$$
  $(\lambda + 1)$   $(\lambda + 2) = \lambda^{3} + 4\lambda^{2} + 5\lambda + 2 = 0$ 

... (12)

Comparing Eq. (12) with Eq. (11), and equating the coefficients of like powers of  $\lambda$  we get:

$$4 = 1.048 + k_3$$
, whence  $k_3 = 2.952 \cdots$  (13)

$$2 = (0.57 + k_1 \ge 0.03)$$
, whence  $k_1 = 47.67 \dots (14)$ 

$$5 = (0.425 + \mathbf{k_1} \times 0.136 + \mathbf{k_2} \times 0.136 + \mathbf{k_3} \times 0.91)$$

or

 $5 = (0.425 + 47.67x \ 0.136 + k_2 x \ 0.136 + 2.952x91 x \ 0.91)$  or

$$5=(0.425 + 6.48 + k_2 \times 0.136 + 2.68)$$
, whence

$$k_2 = -33.71 \cdots \cdots (15)$$

Hence K = 
$$\begin{bmatrix} 0 & 0 & 0 \\ 0 & 0 & 0 \\ k1 & k2 & k3 \end{bmatrix}$$
 =

$$\begin{bmatrix} 0 & 0 & 0 \\ 0 & 0 & 0 \\ 47.67 & -33.71 & +2.952 \end{bmatrix} \dots \dots (16)$$

From Eqn. (9),  $(A - BK) = A_1$ , with shifted poles for Example 1. Or

$$A_1 = \begin{bmatrix} -N_1G_1 & -N_2G_2 & N_3G_3 \\ N_1G_1 & -N_2G_2 & -N_3G_3 \\ -N_1G_1 - k_1 & N_2G_2 - k_2 & -N_3G_3 - k_3 \end{bmatrix} =$$

$$\begin{bmatrix} -0.802 & -0.11 & 0.136 \\ 0.802 & -0.11 & -0.136 \\ -48.472 & 33.6 & -3.088 \end{bmatrix} \cdots \cdots (17)$$

The images/response  $C = \begin{bmatrix} C_1 \\ C_2 \\ C_3 \end{bmatrix}$ ,  $X = \begin{bmatrix} X_1 \\ X_2 \\ X_3 \end{bmatrix}$  in the

autonomous state obtained from digital simulation for  $A_1$  of *Example*, are shown in Fig.11.



Fig. 11: Suppression of Limit Cycles by State Feedback with arbitrarily selection of feedback gain matrix for the *Example*.

# **4.1.2 Optimal Selection of Feedback gain** Matrix using Riccati Equation for Example 1

The Riccati Equation is A'P+PA-PBR<sup>-1</sup>B'P+Q=0

......(18)

And K = Feedback gain matrix = 
$$\mathbb{R}^{-1}$$
 B'P ..... (19)

Assuming R = 1, B= $\begin{bmatrix} 0 \\ 0 \\ 1 \end{bmatrix}$ , Q =  $\begin{bmatrix} 1 & 0 & 0 \\ 0 & 0 & 0 \\ 0 & 0 & 0 \end{bmatrix}$ , A =

$$\begin{bmatrix} -N_1G_1 & -N_2G_2 & N_3G_3 \\ N_1G_1 & -N_2G_2 & -N_3G_3 \\ -N_1G_1 & N_2G_2 & -N_3G_3 \end{bmatrix}$$

Let  $P = \begin{bmatrix} p_{11} & p_{12} & p_{13} \\ p_{21} & p_{22} & p_{23} \\ p_{31} & p_{32} & p_{33} \end{bmatrix}$ , considering P to be

symmetric matrix:  $p_{21}=p_{12}$ ,  $p_{31}=p_{13}$ ,  $p_{32}=p_{23}$ 

Hence P = 
$$\begin{bmatrix} p_{11} & p_{12} & p_{13} \\ p_{12} & p_{22} & p_{23} \\ p_{13} & p_{23} & p_{33} \end{bmatrix}$$

$$\begin{array}{rcrcrc} A' & P & = & \begin{bmatrix} -N_1 G_1 & N_1 G_1 & -N_1 G_1 \\ -N_2 G_2 & -N_2 G_2 & N_2 G_2 \\ N_3 G_3 & -N_3 G_3 & -N_3 G_3 \end{bmatrix} \\ \begin{bmatrix} p_{11} & p_{12} & p_{13} \\ p_{21} & p_{22} & p_{23} \\ p_{31} & p_{32} & p_{33} \end{bmatrix} \\ = & \end{array}$$

$ \begin{bmatrix} (-N_1 G_1 p_{11} + N_1 G_1 p_{21} - N_1 G_2 p_{21}) & (-N_1 G_1 p_{12} + N_1 G_1 p_{21} - N_1 G_2 p_{21}) & (-N_1 G_1 p_{11} + N_1 G_1 p_{22} - N_1 G_2 p_{21}) \\ (-N_2 G_2 p_{11} - N_2 G_2 p_{21} + N_2 G_2 p_{21}) & (-N_2 G_2 p_{12} - N_2 G_2 p_{22} + N_2 G_2 p_{22}) & (-N_2 G_2 p_{21} - N_2 G_2 p_{22} + N_2 G_2 p_{22}) \\ (+N_2 G_2 p_{11} - N_2 G_2 p_{21} - N_2 G_2 p_{21}) & (+N_2 G_2 p_{22} - N_2 G_2 p_{22} - N_2 G_2 p_{22}) & (-N_2 G_2 p_{22} - N_2 G_2 p_{22}) \\ (+N_2 G_2 p_{11} - N_2 G_2 p_{21} - N_2 G_2 p_{21}) & (+N_2 G_2 p_{22} - N_2 G_2 p_{22} - N_2 G_2 p_{22}) & (+N_2 G_2 p_{22} - N_2 G_2 p_{22} - N_2 G_2 p_{22}) \\ \end{bmatrix} $							
(20)							
$ PA = \begin{bmatrix} p_{11} & p_{12} & p_{13} \\ p_{21} & p_{22} & p_{23} \\ p_{31} & p_{32} & p_{33} \end{bmatrix} \begin{bmatrix} -N_1G_1 & -N_2G_2 & N_3G_3 \\ N_1G_1 & -N_2G_2 & -N_3G_3 \\ -N_1G_1 & N_2G_2 & -N_3G_3 \end{bmatrix} = $							
$ \begin{bmatrix} (-N_{1}G_{1}p_{11} + N_{1}G_{2}p_{12} - N_{1}G_{2}p_{11}) & (-N_{2}G_{2}p_{11} - N_{2}G_{2}p_{12} + N_{2}G_{2}p_{13}) & (+N_{4}G_{4}p_{11} - N_{4}G_{4}p_{12} - N_{4}G_{4}p_{13}) \\ (-N_{4}G_{1}p_{21} + N_{4}G_{4}p_{22} - N_{4}G_{4}p_{23}) & (-N_{4}G_{2}p_{21} - N_{4}G_{4}p_{23} + N_{4}G_{2}p_{23}) & (+N_{4}G_{4}p_{21} - N_{4}G_{4}p_{23} - N_{4}G_{4}p_{23}) \\ (-N_{4}G_{4}p_{11} + N_{4}G_{4}p_{22} - N_{4}G_{4}p_{23}) & (-N_{4}G_{4}p_{21} - N_{4}G_{2}p_{23} + N_{4}G_{2}p_{23}) & (+N_{4}G_{4}p_{21} - N_{4}G_{4}p_{23} - N_{4}G_{4}p_{23}) \\ (-N_{4}G_{4}p_{11} + N_{4}G_{4}p_{22} - N_{4}G_{4}p_{23}) & (-N_{4}G_{4}p_{21} - N_{4}G_{2}p_{23} + N_{4}G_{2}p_{23}) & (+N_{4}G_{4}p_{21} - N_{4}G_{4}p_{23} - N_{4}G_{4}p_{23}) \end{bmatrix} $							
(21)							
$PBR^{-1}B'P = \begin{bmatrix} p_{11} & p_{12} & p_{13} \\ p_{21} & p_{22} & p_{23} \\ p_{31} & p_{32} & p_{33} \end{bmatrix} \begin{bmatrix} 0 \\ 0 \\ 1 \end{bmatrix},$							
$\begin{bmatrix} 0 & 0 & 1 \end{bmatrix} \begin{bmatrix} p_{11} & p_{12} & p_{13} \\ p_{21} & p_{22} & p_{23} \\ p_{31} & p_{32} & p_{33} \end{bmatrix} =$							
$ \begin{array}{cccccccccccccccccccccccccccccccccccc$							
$[p_{11}  p_{12}  p_{13}] [0xp_{31}  0xp_{32}  0xp_{33}]$							

0xp<sub>31</sub> 0xp<sub>32</sub> 0xp<sub>33</sub> p<sub>21</sub> p<sub>22</sub> p<sub>23</sub> [p31 p<sub>32</sub> p<sub>33</sub> 1xp<sub>31</sub> 1xp<sub>32</sub> 1xp<sub>33</sub> 0 0 0 [p<sub>11</sub> p<sub>12</sub> P13] 0 0 p<sub>23</sub> 0 p<sub>21</sub> p<sub>22</sub> p31 p<sub>32</sub> p33] p<sub>32</sub> p33 Lp31 p<sub>13</sub> p<sub>31</sub> p<sub>13</sub> p<sub>32</sub> p13 p33 p<sub>23</sub>p<sub>31</sub> p23p32 p23p33 ····· (22)

On substitution of numerical values, Eqn. 20 can be written as

p33p33

 $\begin{bmatrix} (-0.802p_{11}+0.802p_{13}-0.802p_{13}) & (-0.802p_{13}+0.802p_{12}-0.802p_{12}) & (-0.802p_{13}+0.802p_{13}-0.802p_{13}) \\ (-0.11p_{11}-0.11p_{11}+0.11p_{11}+0.11p_{11}) & (-0.11p_{11}-0.11p_{12}+0.11p_{13}) & (-0.11p_{11}-0.11p_{12}+0.11p_{13}) \\ (+0.136p_{12}-0.136p_{13}-0.136p_{13}-0.136p_{13}) & (+0.136p_{12}-0.136p_{12}-0.136p_{13}) & (+0.136p_{13}-0.136p_{13}-0.136p_{13}) \\ \end{bmatrix}$ 

---- (23)

p<sub>33</sub>p<sub>31</sub>

p<sub>33</sub>p<sub>32</sub>

On substitution of numerical values, Eqn. 21 can be written as

 $\begin{bmatrix} (-0.802\rho_{11}+0.802\rho_{12}-0.802\rho_{12}) & (-0.11\rho_{12}-0.11\rho_{12}+0.11\rho_{13}) & (+0.136\rho_{11}-0.136\rho_{12}-0.136\rho_{13}) \\ (-0.802\rho_{21}+0.802\rho_{22}-0.802\rho_{13}) & (-0.11\rho_{22}-0.11\rho_{22}+0.11\rho_{12}) & (+0.136\rho_{21}-0.136\rho_{22}-0.136\rho_{23}) \\ (-0.802\rho_{11}+0.802\rho_{12}-0.802\rho_{13}) & (-0.11\rho_{13}-0.11\rho_{12}+0.11\rho_{13}) & (+0.136\rho_{12}-0.136\rho_{12}-0.136\rho_{13}) \\ ----(24) \end{bmatrix}$ 

On substitution of these values of Eqns. (22), (23), (24) and the assumed value of Q in Riccati Eqn. 18 yields:

 $(-1.604p_{11} + 1.604p_{12} - 1.604p_{13}) + 1 - p^2 13 = 0$ 

$$(-0.912p_{12}+0.802p_{22}-0.802p_{23}-0.11p_{11}+0.11p_{13}-0.01p_{12}+0.01p_{13}-0.01$$

 $p_{13} p_{23}=0$  .....(26)

 $(-0.938 p_{13} + 0.802 p_{23} - 0.802 p_{33} + 0.136 p_{11} - 0.136 p_{12} - 0.002 p_{13} + 0.002 p_{13} - 0.$ 

 $p_{13} p_{33}=0$  .....(27)

 $(\textbf{-0.11}\ p_{13}\textbf{-0.246}\ p_{23}\textbf{+0.11}\ p_{33}\textbf{+0.136}\ p_{12}\textbf{-0.136}\ p_{22}\textbf{-}\ p_{23}$ 

 $p_{33}=0$  .....(29)

 $(0.272 \quad p_{13}-0.272 \quad p_{23}-0.272 \quad p_{33}- \quad p_{33} \quad p_{23}) = 0$ .....(30)

Further, subtracting Eqn. (29) from Eqn. (30), we get,

 $0.382 \; p_{13} - 0.026 \; p_{23} - 0.382 \; p_{33} - 0.136 \; \textbf{p_{12}} + 0.136$ 

 $p_{22}$  .....(31)

The solution of these simultaneous Eqns. (26),(27),(28),(29),(30) & (31) yields :

 $p_{11} = -116.68, p_{12} = -110.48, p_{13}=6.58, p_{22} = -93.24, p_{23} = -6.58, p_{33} = 0$ 

From Eqn. (19),  $K = R^{-1} B'P = 1$  [001]  $\begin{bmatrix} p_{11} & p_{12} & p_{13} \\ p_{12} & p_{22} & p_{23} \\ p_{13} & p_{23} & p_{33} \end{bmatrix}$   $Or[k_1k_2k_3] =$ 

 $[(0xp_{11} + 0xp_{12} + 1xp_{13}) (0xp_{12} + 0xp_{22} + 1xp_{23}) (0xp_{13} + 0xp_{23} + 1xp_{33})]$ 

Or  $[k_1k_2k_3] = [(p_{13})(p_{23})(p_{33})]$ 

Whence,  $k_1 = 6.58$ ,  $k_2 = -6.58$  and  $k_3 = 0$ 

••••• (32)

Hence,  $A - BK = A_2 =$ 



Fig. 12: Suppression of Limit Cycles by State Feedback with optimal selection of feedback gain matrix for the *Example*.

On substitution of numerical values for Example 1,  $A_2$  becomes:

$$A_{2}= \begin{bmatrix} -0.802 & -0.11 & 0.136\\ 0.802 & -0.11 & -0.136\\ -802 - 6.58 & 0.11 + 6.5 & -0.136 - 0 \end{bmatrix} = \begin{bmatrix} -0.802 & -0.11 & 0.136\\ 0.802 & -0.11 & -0.136\\ -7.382 & 6.69 & -0.136 \end{bmatrix} \dots \dots (33)$$

The images/responses  $C = \begin{bmatrix} C_1 \\ C_2 \\ C_3 \end{bmatrix}$  and  $X = \begin{bmatrix} X_1 \\ X_2 \\ X_3 \end{bmatrix}$  in the

autonomous state, obtained from digital simulation for the *Example*, are shown in Fig. 12.

# **5** Conclusion

In today's scenario, nonlinear self-sustained oscillations or Limit Cycles are the basic feature of instability. The existence /exhibition of such phenomena limit the performance of most of the physical systems such as the speed and position control in robotics, automation industry in particular. Ouenching/Complete extinction of such LC has been a severe headache among the researchers for several decades. There are some methods, seen in the available literature which suggests the solution to this problem occurring in SISO or 2x2 systems. However, our present work explores the solution for 3x3 systems in the event of existence of LC problem and establishes the result graphically & validated by digital simulation. The novelty of the work claims in: (i) Quenching of LC exhibited in nonlinear systems by Signal Stabilization with deterministic as well as random (Gaussian) signals, (ii) Suppression of limit cycles in 3x3 nonlinear systems by Pole Placement using State feedback with arbitrary selection as well as optimal selection of feedback gain matrix K.

More importantly the poles of such 3x3 systems are shifted or placed suitably by State feedback so that the system do not exhibit limit cycles. This pole placement is done either by arbitrary selection satisfying the complete state controllability condition or by optimal selection of feedback gain matrix K using Riccati equation *which has not been attempted elsewhere*.

The present work has the brighter future scope of adopting the techniques like signal stabilization [44] and suppression of limit cycles [46] in the event of the existence of limit cycling oscillations for 3x3 higher dimensional systems through an exhaustive analysis.

Analytical/Mathematical procedures may also be developed for signal stabilization using both deterministic and random signals applying DIDF and RIDF respectively.

Backlash is one of the nonlinearities commonly occurring in physical systems which are an inherent characteristic of Governor, more popularly used for load frequency control (LFC) in power systems. The LFC shows poor performance due to the backlash characteristic of the governor. Similarly, the backlash characteristic limits the performance of speed and position control in the robotics, automation industry. The poor performance of LFC, speed and position control in robotics and in automation industries are happening since these systems exhibit limit cycles due to their backlash type of nonlinear characteristics. The proposed method of suppression of L.C. can be extended and developed for backlash type nonlinearity in 3x3 systems and used to completely eliminate the limit cycle to mitigate such problems.

The phenomena of synchronization and desynchronization can be observed/identified analytically using Incremental Input Describing function (IDF).

# Acknowledgement:

The Authors wish to thank the C.V Raman Global University, Bhubaneswar – 752054, Odisha, India for providing the computer facilities for carrying out the research and preparation of this paper.

#### References:

- [1] Gelb, A, Limit cycles in symmetric multiple nonlinear systems. *IEEE Trans. Autumn. Control*: AC-8, 1963, pp. 177-178.
- [2] Jud, H.G Limit cycle determination of parallel linear and non- linear elements. *IEEE Trans. Autumn. Control*: AC-9, 1964, pp. 183-184.
- [3] Gran, R., and Rimer, M Stability analysis of systems with multiple nonlinearities. *IEEE Trans. Autumn. Control:* 10, 1965, pp. 94-97.
- [4] Davison, E.J., and Constantinescu, D Describing function technique for multiple nonlinearity in a single feedback system *IEEE Trans Autumn*. *Control:* AC-16: 1971, pp. 50-60
- [5] Oldenburger, R., T. Nakada T Signal stabilisation of self - oscillating system *IRE Trans. Automat Control.* USA, 6, 1961, pp: 319-325.
- [6] Patra, K. C, Patnaik, A, Investigation of the Existence of Limit Cycles in Multi Variable Nonlinear Systems with Special Attention to 3x3 Systems. Int. Journal of Applied Mathematics, Computational Science and System Engineering. Vol. 5, 2023, pp. 93-114.

- [7] Nordin, M. and Gutman, P. O Controlling mechanical systems with backlash- a survey, *Automatica*, vol. 38, (10), 2002, pp.1633-1649.
- [8] Wang, C., Yang, M., Zheng, W., Hu, K. and Xu, D, Analysis and suppression of limit cycle oscillation for Transmission System with backlash Nonlinearity, *IEEE Transactions on Industrial Electronics*, vol. 62, (12), 2017, pp. 9261-9270.
- [9] Viswandham, N., and Deekshatulu, B.L Stability analysis of nonlinear multivariable systems. *Int. J. Control*, 5, 1966, pp. 369-375.
- [10] Gelb, A. and Vader-Velde, W.E Multiple-input describing functions and nonlinear system design, McGraw-Hill, New York, 1968
- [11] Nikiforuk, P.N., and Wintonyk, B.L.M Frequency response analysis of two-dimensional nonlinear symmetrical and non-symmetrical control systems. *Int. J. Control*, 7, 1968, pp.49-62.
- [12] Raju, G.S., and Josselson, R Stability of reactor control systems in coupled core reactors, *IEEE Trans. Nuclear Science*, NS-18, 1971, pp. 388-394.
- [13] Atherton, D.P Non-linear control engineering -Describing function analysis and design. Van Noslrand Reinhold, London, 1975
- [14] Atherton, D.P., and Dorrah, H.T A survey on nonlinear oscillations, *Int. J. Control*, 31. (6), 1980, pp. 1041-1105.
- [15] Gray, J. O. And Nakhala, N.B Prediction of limit cycles in multivariable nonlinear systems. *Proc. IEE*, *Part-D*, 128, 1981 pp. 233-241.
- [16] Mees, A.I Describing function: Ten years on. IMA J. Appl. Math., 34, 1984 pp. 221-233.
- [17] Sebastian, L the self-oscillation determination to a category of nonlinear closed loop systems, *IEEE Trans. Autumn. Control*, AC-30, (7), 1985 pp. 700-704.
- [18] Cook, P.A, Nonlinear dynamical systems, Prentice-Hall, Englewood ClilTs, NJ, 1986
- [19] Chang, H.C., Pan, C.T., Huang, C.L., and Wei, C.C A general approach for constructing the limit cycle loci of multiple nonlinearity systems, *IEEE Trans. Autumn. Control*, AC-32, (9), 1987, pp. 845-848.
- [20] Parlos, A.G., Henry, A.F., Schweppe, F.C., Gould, L.A., and Lanning, D.D Nonlinear multivariable control of nuclear power plants based on the unknown but bounded disturbance model, *IEEE Trans. Autumn. Control*, AC-33, (2), 1988 pp. 130-134.
- [21] Pillai, V.K., and Nelson, H.D A new algorithm for limit cycle analysis of nonlinear systems, *Trans. ASME, J. Dyn. Syst. Meas. Control*, 110, 1988, pp. 272-277.
- [22] Genesio, R., and Tesi, A On limit cycles of feedback polynomial systems, *IEEE Trans. Circuits Syst.*, 35, (12), 1988, pp. 1523-1528.
- [23] Fendrich, O.R Describing functions and limit cycles, *IEEE Trans. Autom. Control*, AC -31, (4), 1992, pp. 486487.

- [24] Zhuang, M., and Artherton, D.P PID controller design lor TITO system, *TEE Proc. Control Theory Appl.* 141, (2), 1994, pp. 111-120.
- [25] Loh, A.P., and Vasanu, V.V Necessary conditions for limit cycles in multi loop relay systems, *IEE Proc.*, *Control Theory Appl.*, 141, 31, 1994, pp. 163-168.
- [26] Hakimi, A. R. and Binazadeh, T, Inducing sustained oscillations in a class of nonlinear discrete time systems, *Journal of Vibration and control* vol. 24, Issue 6, July, 20, 2016.
- [27] Tesi, A, Abed, E. H., Genesio, R., Wang, H. O., Harmonic balance analysis of periodic doubling bifurcations with implications for control of nonlinear dynamics, Automatic, 32 (9), 1996, 1255, 1271.
- [28] Habib, G, and Kerschen, G. Suppression of limit cycle oscillations using the nonlinear tuned vibration absorber. *Mathematical Physical and Engineering Sciences*, 08 April 2015 https://dol.org
- [29] Lim, L. H and Loh, A.P. Forced and sub-harmonic oscillations in relay feedback systems, *Journal of* the Institution of Engineers Singapore, 45(5),(2005),pp88-100
- [30] Hori, Y., Sawada, H., Chun, Y., Slow resonance ratio control for vibration suppression and disturbance rejection in torsional system, *IEEE Trans. Ind. Electron.*, vol. 46, (1), 1999, pp.162-168.
- [31] Raj Gopalan, P.K and Singh, Y. P. Analysis of harmonics and almost periodic oscillations in forced self-oscillating systems, *Proc* 4<sup>th</sup> IFAC Congress, Warsaw.41,(1969),80-122
- [32] Lin, C.H., Han, K.W Prediction of Limit cycle in Nonlinear two input two output control system, '*IEE Proc.-Control Theory Appl.* Vol.146, No.3 may. 1999.
- [33] Chidambaram, I.A, and Velusami, S Decentralized biased controllers for load-frequency control of inter connected power systems considering governor dead band non-linearity, INDICON, Annual *IEEE*, 2005, pp.521-525.
- [34] Eftekhari, M and Katebi, S. D Evolutionary Search for Limit Cycle and Controller Design in Multivariable nonlinear systems, *Asian Journal of Control*, Vol. 8, No. 4, 2006, pp. 345 – 358.
- [35] Katebi, M., Tawfik, H., Katebi, S. D., Limit Cycle Prediction Based on Evolutionary Multi objective Formulation, *Hindawi Publishing Corporation*, Mathematical Problems in engineering Volume, Article ID 816707, 2009, 17pgs.
- [36] Garrido, J, Morilla, F., Vazquez, F., Centralized PID control by Decoupling of a Boiler-Turbine Unit, *Proceedings of the European Control Conference*, Budapest, Hungary, Aug. 2009, 23-26.
- [37] Tsay, T.S Load Frequency control of interconnected power system with governor backlash nonlinearities, *Electrical Power and Energy*, vol. 33, 2011, pp.1542-1549.

- [38] Tsay, T.S Limit Cycle prediction of nonlinear multivariable feedback control systems with large transportation lags, *Hindawi Publishing corporation journal of control science and Engineering*, Vol., article id 169848, 2011.
- [39] Tsay, T.S Stability Analysis of Nonlinear Multivariable Feedback Control systems, WSEAS Transactions on systems, Volume 11, Issue 4, 2012, pp. 140 – 151.
- [40] Sujatha, V., Panda, R. C Relay Feedback Based Time domain modelling of Linear 3-by-3 MIMO System, American Journal of System Science, Scientific & Academic Publishing, 1(2) 2012, pp. 17-22.
- [41] Wang, C, Ming, Y, Weilong, Z., Jiang, L., and Dianguo, X., Vibration suppression with shaft torque limitation using explicit MPC-PI switching control in elastic drive systems, *IEEE Trans. Ind. Electron*, vol. 62, (11), 2015, pp. 6855-6867.
- [42] Yang, M, Weilong, Z., Jiang, L. and Dianguo, X., Suppression of mechanical resonance using torque disturbance observer for two inertia system with backlash *Proc. IEEE 9th Int. Conf. Power Electron.*, ECCE Asia, 2015, pp. 1860 - 1866.
- [43] Shi, Z, and Zuo, Z back stepping control for gear transmission servo systems with backlash nonlinearity *IEEE Trans. Autumn. Sci. Eng.*, vol. 12, (2), 2015, pp. 752-757.
- [44] Patra, K. C, and Dakua, B. K, Investigation of limit cycles and signal stabilisation of two dimensional systems with memory type nonlinear elements, *Archives of Control Sciences*, vol. 28, (2), 2018, pp. 285-330.
- [45] Zeineb, R., Chekib, G. and Naceur, B. B Non-fragile

*H*<sub>∞</sub> Stabilizing Nonlinear Systems Described by Multivariable Hammerstein Models *Nonlinear Dynamics of Complex Systems, Hindawi* (Special Issue) Volume 2021,19 Feb. 2021,

- [46] Patra, K. C, Kar, N Suppression Limit cycles in 2 x 2 nonlinear systems with memory type nonlinearities, *International Journal of Dynamics and Control*, Springer Nature', 34,95€, vol.10 Issue 3, 2022, pp 721-733.
- [47] Elisabeth, T.M & Seng, C. C. Designing Limit-Cycle Suppressor Using Dithering and Dual-Input Describing Function Methods. *Mathematics*, *Vol.8(MDP1) No.6,2020*
- [48] Keran, S, Xiaolong, W and Rongwei, G. Stabilization of Nonlinear Systems with External Disturbances Using the DE-Based Control Method Symmetry (MDPI), 15, 987,2023
- [49] Stanislaw, H. Zak Systems and Control' Oxford University Press, 2003, pp. 77 – 83.
- [50] Ogata K, Modern control engineering, 5<sup>th</sup> Edn. P H I Learning, pp. 723-724 and 2012.
- [51] Raymond, T., Shahian, B., JR. C. J. S., and Hostetter, G. H., Design of Feedback Control

*Systems*, Oxford University Press, 4<sup>th</sup> edition, 2002, pp. 677-678.

- [52] Patra, K.C, Kar, N, Signal Stabilization of Limit cycling two Dimensional Memory Type Nonlinear Systems by Gaussian Random Signal, International Journal of Emerging Trends & Technology in Computer Science, Vol.9 Issue 1,2020, PP.10-17
- [53] Patra, K.C., Pati, B.B., Kacprzyk, J., Prediction of limit cycles in nonlinear multivariable systems, Arch. Control Sci. Poland, 4(XL) 1995, pp 281-297.

# **Contribution of Individual Authors to the Creation of a Scientific Article (Ghostwriting Policy)**

Kartik Chandra Patra has formulated the problem, methodology of analysis adopted and algorithm of computation presented.

Asutosh Patnaik has made the validation of the results using the geometric tools and SIMULINK toolbox of MATLAB software.

#### Sources of Funding for Research Presented in a Scientific Article or Scientific Article Itself

The C.V. Raman Global University has provided all computer facilities with relevant software for the research work and also for the preparation of the paper.

#### **Conflict of Interest**

The authors have no conflicts of interest to declare that are relevant to the content of this article.

# Creative Commons Attribution License 4.0 (Attribution 4.0 International, CC BY 4.0)

This article is published under the terms of the Creative Commons Attribution License 4.0

https://creativecommons.org/licenses/by/4.0/deed.en \_US